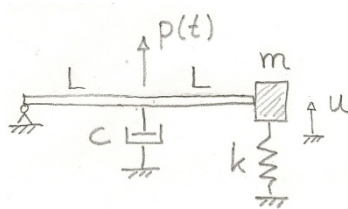


Theory exam in Structural Dynamics 2013-03-05 kl. 10-12

The test consists of 6 questions giving the maximum of 15 points. Each question should be answered on a separate paper. No helping aids are permitted on this test, except calculator. Do not forget to write your name on each submitted paper.

1) (2 p)

A weightless rigid bar is connected to a discrete mass in one end and a frictionless joint in the other end according to the figure.



Show by using free body diagrams that the equation of motion is

$$m\ddot{u} + \frac{c}{4}\dot{u} + ku = \frac{1}{2}p(t)$$

2) (2 p)

Determine the steady state response $u(t)$ for $p(t) = p_0 \sin(\omega t)$ at resonance for the system in 1) if $c/4m = 2\zeta\omega_n$ and $\zeta = 0.1$. Use complex analysis or the deformation response factor as shown in Appendix to determine the vibration amplitude u_0 .

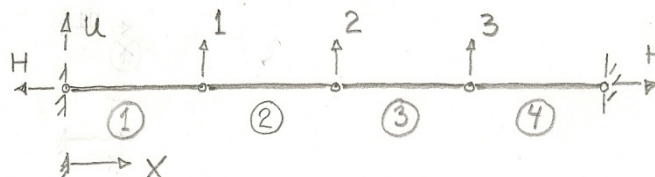
3) (2 p)

A system with N dofs is given in general terms as $m\ddot{u} + ku = p(t)$ and the full modal expansion is given by $u = \Phi q = \sum \phi_n q_n$.

- Establish the uncoupled system of equations given by $M\ddot{q} + Kq = P(t)$ i.e. determine the matrices M , K , and the vector P .
- Determine initial values for the modal coordinates i.e. determine $q(0)$ and $\dot{q}(0)$ in terms of $u(0)$ and $\dot{u}(0)$.
- A solution $q(t)$ has been found. How are the physical displacements determined?

4) (3p)

A model of a string is established by using four linear elements with mass and stiffness matrices given below. The mass matrix can be either lumped or consistent.



The element matrices are given by: $\frac{m}{2} \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, or $\frac{m}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$, and $k \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$ ($k = H/L_e$)

a) Show that the assembled system matrices becomes

$$\mathbf{m}^l = m \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad \text{or} \quad \mathbf{m}^c = \frac{m}{6} \begin{bmatrix} 4 & 1 & 0 \\ 1 & 4 & 1 \\ 0 & 1 & 4 \end{bmatrix}, \quad \text{and} \quad \mathbf{k} = k \begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 2 \end{bmatrix}$$

b) A damping matrix is introduced in the case of the consistent mass matrix as

$$\mathbf{c} = c \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}. \quad \text{Is this classical damping? Motivate your answer.}$$

5) (3p)

Consider the system (without damping) given in question 4), in the case of the lumped mass matrix. The eigen-frequencies and vectors are

$$\omega_1^2 = (2 - \sqrt{2}) \frac{k}{m}, \quad \omega_2^2 = 2 \frac{k}{m}, \quad \text{and} \quad \omega_3^2 = (2 + \sqrt{2}) \frac{k}{m}$$

$$\boldsymbol{\phi}_1 = \begin{bmatrix} 1 \\ \sqrt{2} \\ 1 \end{bmatrix}, \quad \boldsymbol{\phi}_2 = \begin{bmatrix} 1 \\ 0 \\ -1 \end{bmatrix}, \quad \text{and} \quad \boldsymbol{\phi}_3 = \begin{bmatrix} -1 \\ \sqrt{2} \\ -1 \end{bmatrix}$$

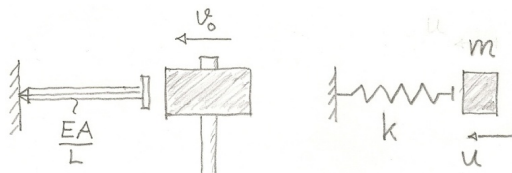
a) Use a linear deflection $\boldsymbol{\Psi}_1 = \begin{bmatrix} 1 \\ 2 \\ 1 \end{bmatrix}$ to determine an estimate of the first eigen-frequency for the system above.

b) Use also a second vector $\boldsymbol{\Psi}_2 = \begin{bmatrix} 1 \\ -1 \\ 1 \end{bmatrix}$ to set up a reduced system. Answer with the reduced systems matrices $\tilde{\mathbf{m}}$ and $\tilde{\mathbf{k}}$.

c) The eigen-frequencies of the reduced system are $\tilde{\omega}_1^2 = 0.586 \frac{k}{m}$, and $\tilde{\omega}_2^2 = 3.41 \frac{k}{m}$. Compare with the exact frequencies and vectors. Give comments on the comparison.

6) (3 p)

A nail and a hammer are modeled by a linear spring and an incident mass according to the figure. (The nail is supposed to be driven into a piece of very hard wood and cannot move into the wood for the velocity given to the hammer.)



The hammer has an initial velocity $v_0=5\text{m/s}$ and a mass $m=0.7\text{kg}$. The nail is made of steel $E=210\text{GPa}$ and the cross section is square $A=2\times 2\text{mm}$, with a length $L=0.1\text{m}$.

Determine the maximum force F_{max} reached in the contact between hammer and nail and the duration Δt of the force pulse.

Appendix:

